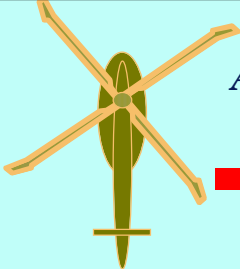




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**Alfred Gessow Rotorcraft Center**




UNIVERSITY OF MARYLAND



**Aeroelastic Modeling of Trailing-Edge Flap Rotor With Smart Materials Actuation**

**Jinwei Shen**      **Inderjit Chopra**  
*Graduate Student*      *Professor & Director*

**Overview**



**1. Introduction**      - *Objective and Scope of Present Research*

**2. Analytic Model**      - *Structural model*  
   - *Coupled blade-flap-actuator model*

**3. Analytic Results**      - *Correlation with CAMRAD II*  
   - *Parametric study*

**4. Summary and Conclusions**

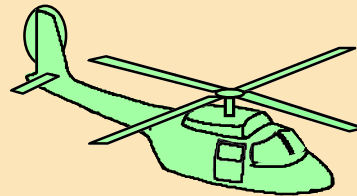
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# UMARC

(University of Maryland Advanced Rotorcraft Code)



- Finite element methods in space and time
- Nonlinear elastic blade with moderate deflections
- Unsteady flap aerodynamics (Leishman-Hariharan)
- Free wake (Bagai-Leishman)
- Modal reduction
- Coupled trim solution for free flight and wind tunnel conditions



# Introduction



**Old UMARC Model: Trailing-edge flap motion was prescribed.**

**New Model: Blade-flap-actuator as a coupled system, trailing-edge flap motion is calculated along with the blade response.**



Smart actuation flap



Analytical model

Hinge Spring



## Objective



**Develop a comprehensive rotorcraft aeroelastic analysis with fully coupled blade-flap-actuator system, and study the effect of this coupling on vibration minimization with trailing edge flap.**

Carry out

- Correlation with CAMRAD II predictions
- Parametric studies with coupled blade-flap-actuator model for MD900 rotor



MD900 Explorer: 5-bladed bearingless rotor

## Analytical Model



**Coupled blade-flap-actuator equations:**

$$\begin{aligned} \text{Blade} &\rightarrow \begin{bmatrix} M_{bb} & M_{bf} \\ M_{fb} & M_{ff} \end{bmatrix} \begin{Bmatrix} \ddot{q}_b \\ \ddot{\delta} \end{Bmatrix} + \begin{bmatrix} C_{bb} & C_{bf} \\ C_{fb} & C_{ff} \end{bmatrix} \begin{Bmatrix} \dot{q}_b \\ \dot{\delta} \end{Bmatrix} \\ \text{Flap} &\rightarrow \begin{bmatrix} K_{bb} & K_{bf} \\ K_{fb} & K_{ff} \end{bmatrix} \begin{Bmatrix} q_b \\ \delta \end{Bmatrix} = \begin{Bmatrix} F_b \\ F_f \end{Bmatrix} \end{aligned}$$

**Trailing-edge flap equation contains actuator dynamics**

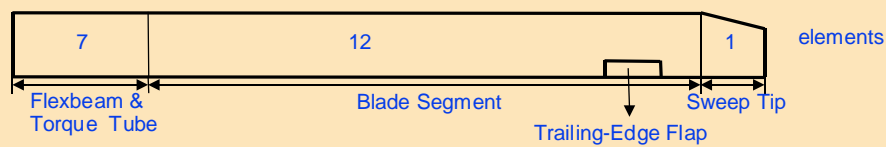
$q_b$  Blade displacement vector

$\delta$  Flap angle

## Structural Modeling



• <b>Blade modeling</b>	- Flexbeam and torque tube	7
	- Blade segment	12
	- Sweep tip	1
• <b>Modal analysis</b>	- Blade modes	10
• <b>Flap configuration</b>	- Flap Chord	25%
	- Flap length	18%
	- Flap location	83%
• <b>Wind tunnel trim condition</b>	- Advancing ratio	0.2
	- Thrust coefficient ( $C_T/\sigma$ )	0.0774
	- Shaft angle (backward)	- 3.5



## Coupled Blade-Flap-Actuator



- **Trailing edge flap response**
- **Actuator stiffness**
- **Unlocked and locked flap**

# Trailing-Edge Flap Angle

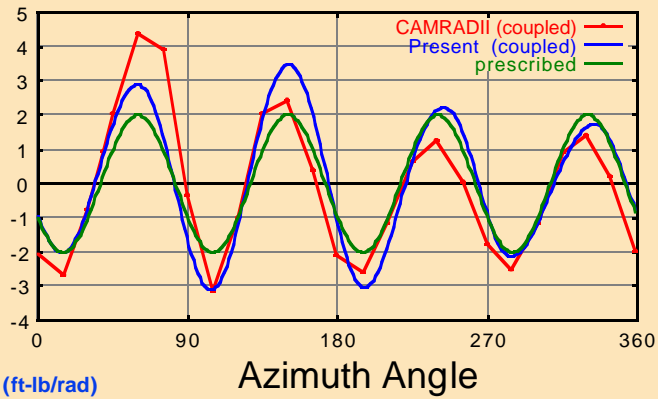


- $\mu = 0.20$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R Location = 83%R

Input:  $\delta_f = 2 \cos(4\psi - 240^\circ)$

Flap Angle (deg)



- Hinge Spring = 81.52 (ft-lb/rad)
- Hinge damping = 0.05 (ft-lb/rad/sec)
- Flap effectiveness factor = 1

# Trailing-Edge Flap Hinge Shear

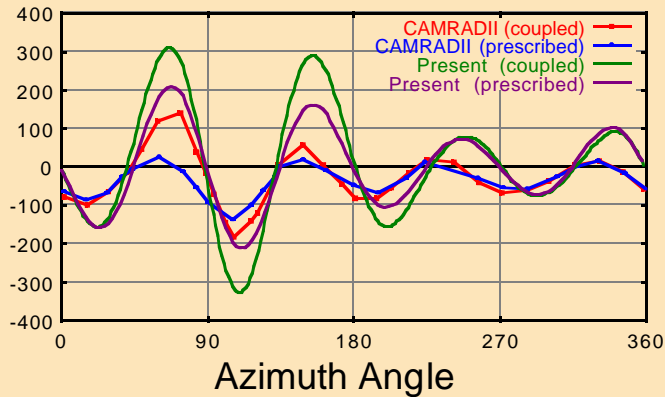


- $\mu = 0.20$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R Location = 83%R

Input:  $\delta_f = 2 \cos(4\psi - 240^\circ)$

Trailing-Edge Flap Hinge Shear (lb)



## Trailing-Edge Flap Hinge Moment

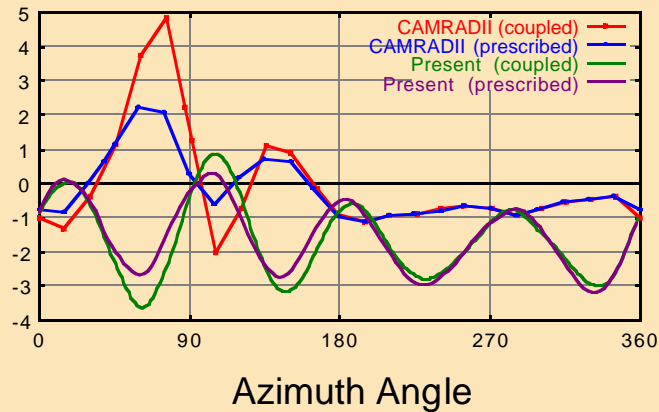


- $\mu = 0.20$
- $c_{\delta}/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R Location = 83%R

Input:  $\delta_f = 2 \cos(4\psi - 240^\circ)$

Trailing-Edge  
Flap Hinge  
Moment  
(ft-lb)



Discrepancy possible because of aerodynamic forces

## Actuator Stiffness Effect

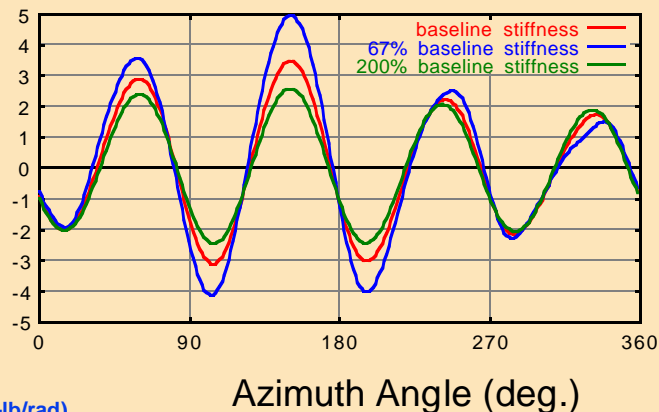


- $\mu = 0.20$
- $C_T/\sigma = 0.0774$
- $l_{\delta}/R = 18\%$
- $r_{\delta}/c = 83\%$
- $c_{\delta}/c = 25\%$

Input:  $\delta_f = 2 \cos(4\psi - 240^\circ)$

Flap Response

Flap  
Angle  
(deg.)



- Hinge Spring = 81.52 (ft-lb/rad)
- Hinge damping = 0.05 (ft-lb/rad/sec)

# Unlocked Flap

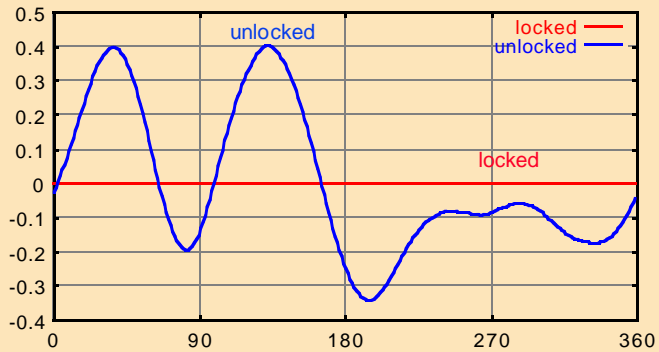


- $\mu = 0.20$
- $C_T/\sigma = 0.0774$
- $l_f/R = 18\%$
- $r_f/c = 83\%$
- $c_f/c = 25\%$

Input:  $\delta_f = 0$  (No Actuator Force)

## Flap Response

Flap Angle (deg.)



Azimuth Angle (deg.)

- Peak-to-Peak flap amplitude: 0.75 deg.
- Large flap response at azimuth angle 0-180 (deg)

# Unlocked Flap

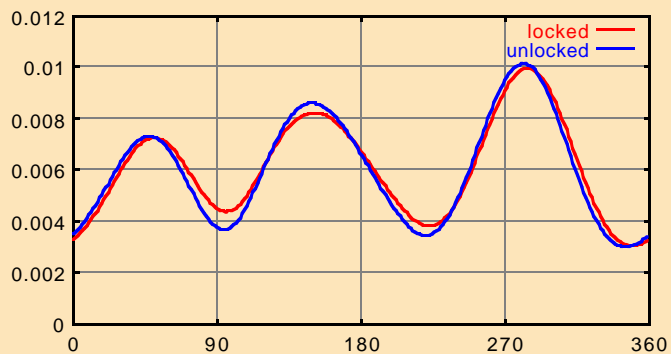


- $\mu = 0.20$
- $C_T/\sigma = 0.0774$
- $l_f/R = 18\%$
- $r_f/c = 83\%$
- $c_f/c = 25\%$

Input:  $\delta_f = 0$  (No Actuator Force)

## Blade Tip Flap Response

Blade Tip Flap Response (w/R)



Azimuth Angle (deg.)

- Peak-to-Peak tip amplitude increase: 5%

# Unlocked Flap

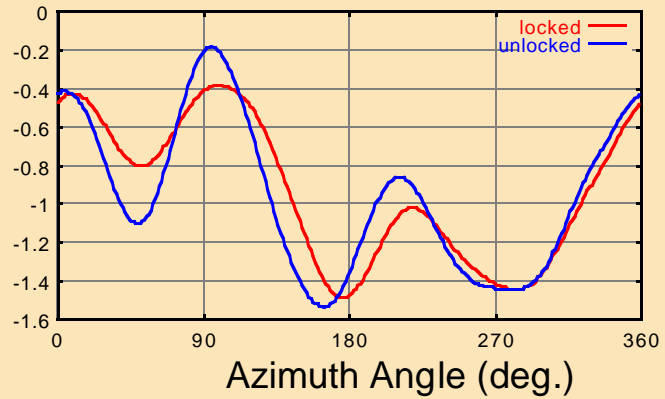


- $\mu = 0.20$
- $C_T/\sigma = 0.0774$
- $l/R = 18\%$
- $r/c = 83\%$
- $c/c = 25\%$

Input:  $\delta_f = 0$  (No Actuator Force)

## Blade Tip Torsion Response

Blade Tip Torsion Response (deg.)



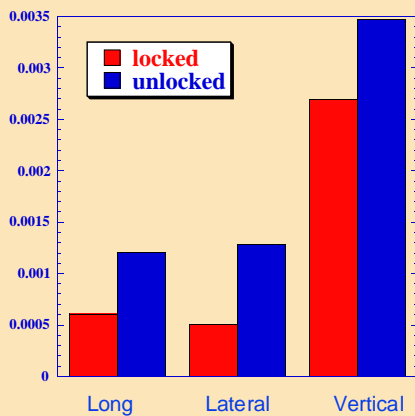
• Peak-to-Peak tip amplitude increase: 15%

# Unlocked Flap

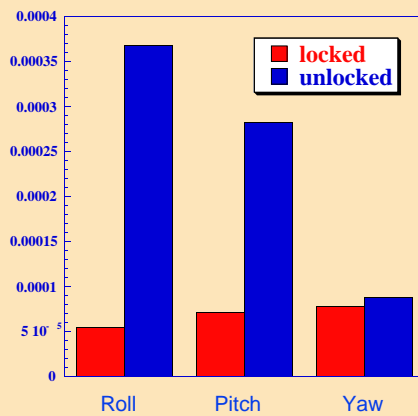
Input:  $\delta_f = 0$



## 5/Rev Hub Forces



## 5/Rev Hub Moments



Uncontrolled hub forces increase due to flap/actuator dynamics

## Parametric Study (with coupled system)



- **Design Parameters**
  - Flap span-wise location
  - Flap span-wise size
  - Blade pitch link stiffness
  - Advancing ratio
- **Trend Studies**
  - Objective function ( 5/rev hub loads)
  - Optimal flap peak-to-peak angle
  - Actuation power

Study coupling effect due to blade-flap-actuator dynamics

## Multicyclic controller



- **Objective function**

$$J \equiv Z_n^T W_z Z_n + \theta_n^T W_\theta \theta_n$$

- **Local linearization**

$$Z_n = Z_{n-1} + T(\theta_n - \theta_{n-1})$$

Z : 5/Rev hub loads  
(3 forces + 2 moments)  
W: Weighting matrix  
θ : 4,5 and 6/Rev flap angles  
T : Transfer function

- **Weighting matrix (diagonal)**

$$W_z = \begin{bmatrix} 0.040 & 0.023 & 1.000 & 0.440 & 0.136 \end{bmatrix}$$

Long
Lateral
Vertical
Roll
Pitch

Controller Works for both coupled system and prescribed system

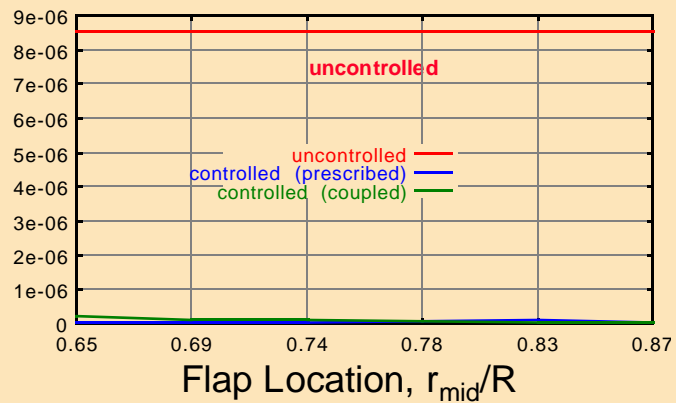
## Effect of Flap Location



- $\mu = 0.30$
- $c_d/c = 25\%$
- $C_T/\sigma = 0.0774$

Vibration Objective Function  $J_0$  and  $J$

### Flap = 18%R Hub Loads (3 forces + 2 moments)



- Complete minimization with prescribed and coupled flap

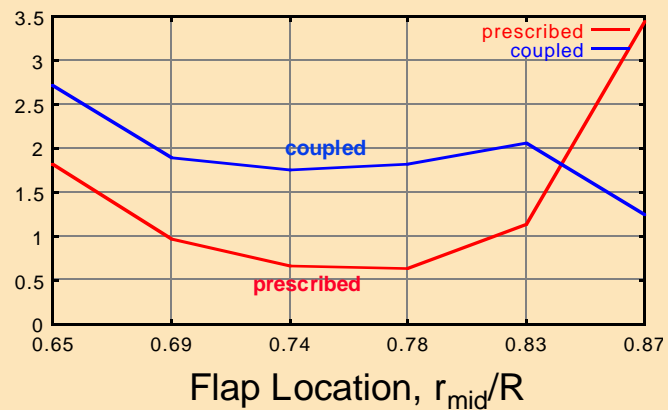
## Effect of Flap Location



- $\mu = 0.30$
- $c_d/c = 25\%$
- $C_T/\sigma = 0.0774$

Peak-to-Peak Flap Deflection  $\|\delta\|$  (deg.)

### Flap = 18%R Flap Deflections



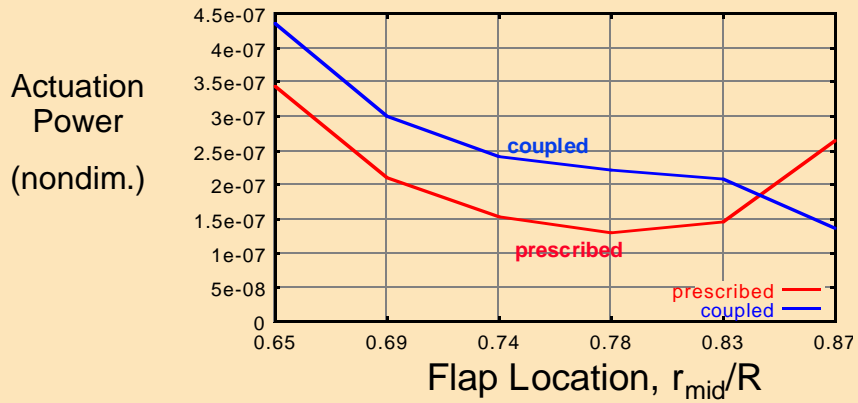
- Peak-to-peak flap deflection for coupled system increase: 100%

## Effect of Flap Location



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

### Flap = 18%R Actuation Power



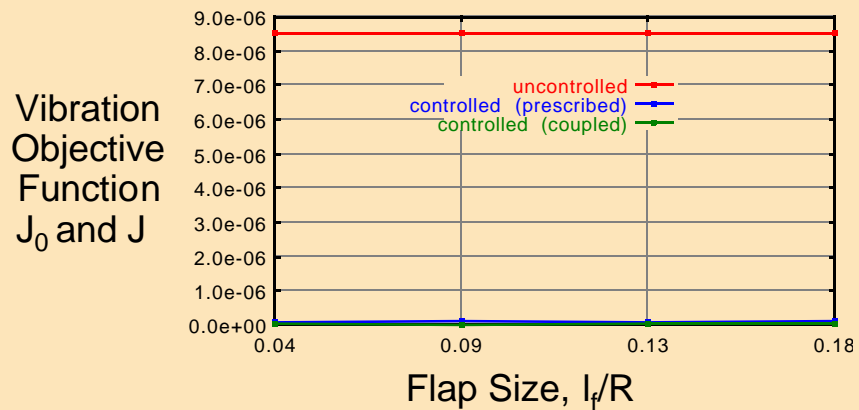
- Power for coupled system increase: 40%

## Effect of Flap Size



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

### Flap location = 83%R Hub Loads (3 forces + 2 moments)



- Complete minimization with prescribed and coupled flap

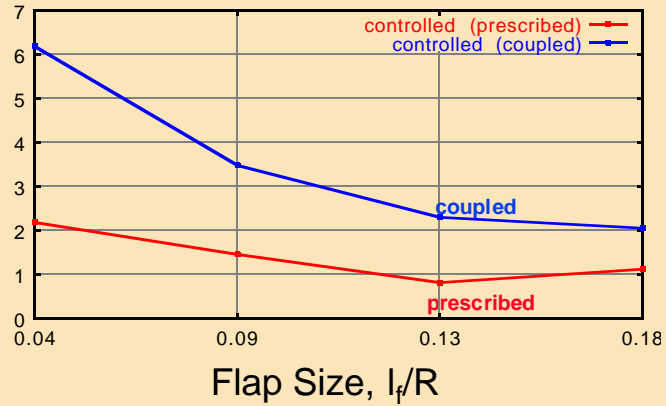
## Effect of Flap Size



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

### Flap location = 83%R Flap Deflections

Peak-to-Peak  
Flap  
Deflection  
 $||\delta||$   
(deg.)



- Peak-to-peak flap deflection for coupled system increase: 200%

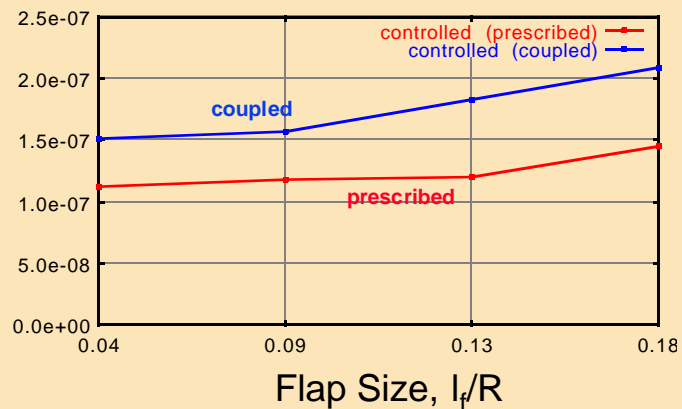
## Effect of Flap Size



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

### Flap location = 83%R Actuation Power

Actuation  
Power  
(nondim.)



- Power for coupled system increase: 40%

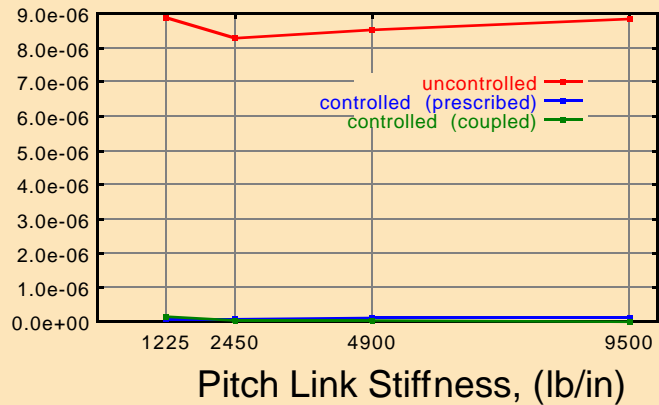
## Effect of Pitch Link Stiffness



- $\mu = 0.30$
- $c_{\phi}/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R location = 83%R  
Hub Loads (3 forces + 2 moments)

Vibration Objective Function  $J_0$  and J



- Complete minimization with prescribed and coupled flap

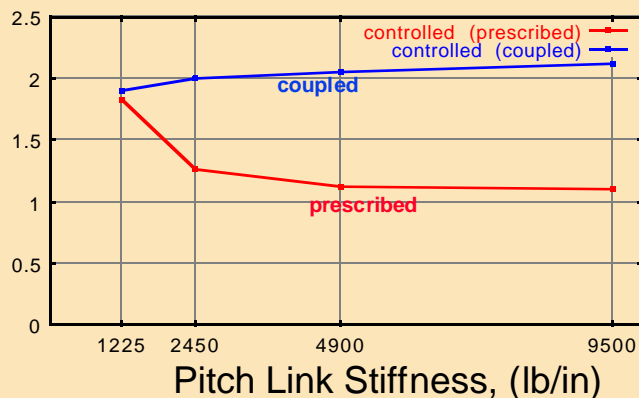
## Effect of Pitch Link Stiffness



- $\mu = 0.30$
- $c_{\phi}/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R location = 83%R  
Flap Deflections

Peak-to-Peak Flap Deflection  $||\delta||$  (deg.)



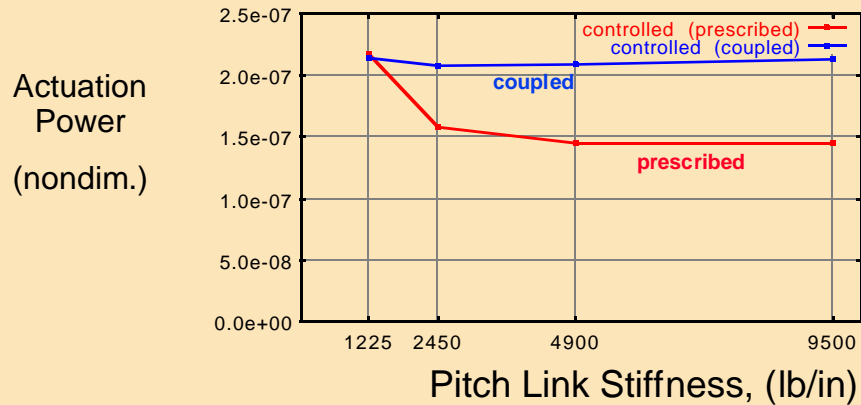
- Peak-to-peak flap deflection for coupled system increase: 90%

## Effect of Pitch Link Stiffness



- $\mu = 0.30$
- $c_{\phi}/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R location = 83%R  
Actuation Power



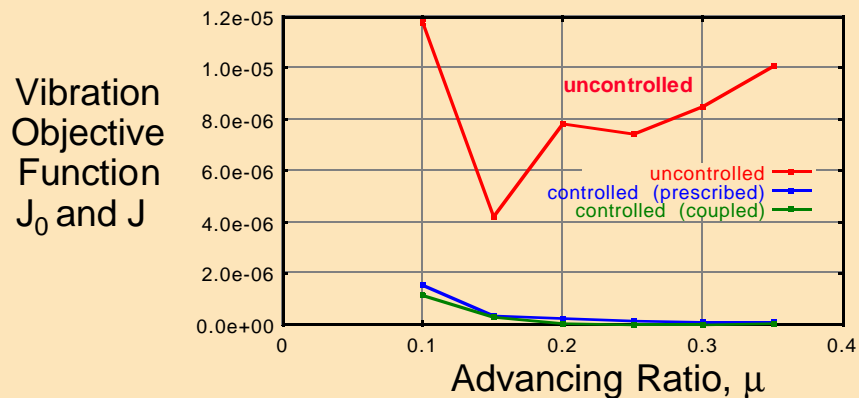
- Power for coupled system increase: 30%

## Forward Speed Effect



- $\mu = 0.30$
- $c_{\phi}/c = 25\%$
- $C_T/\sigma = 0.0774$

Flap = 18%R  
Hub Loads (3 forces + 2 moments)



- Vibration objective function was minimized to less than 10%

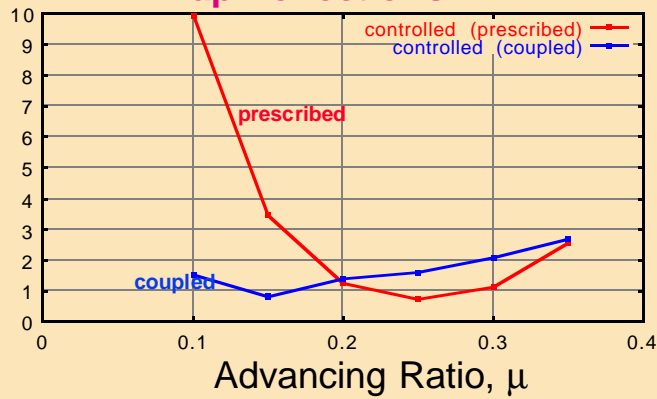
## Forward Speed Effect



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

Peak-to-Peak  
Flap Deflection  
 $||\delta||$   
(deg.)

### Flap = 18%R Flap Deflections



- For prescribed system, peak-to-peak flap deflection: 0.8 degree to 10 degree
- For coupled system, peak-to-peak flap deflection: 0.9 degree to 2.6 degree

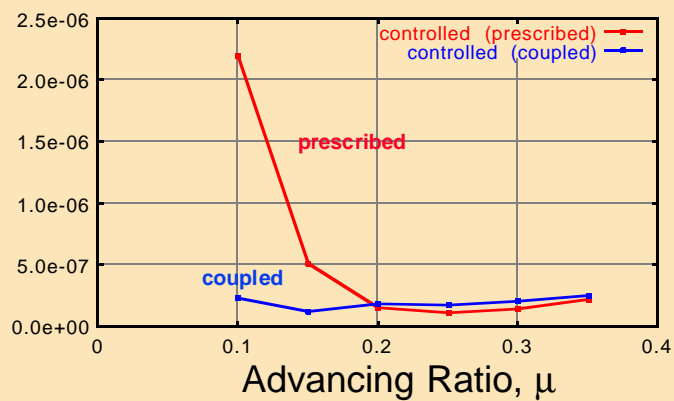
## Forward Speed Effect



- $\mu = 0.30$
- $c_f/c = 25\%$
- $C_T/\sigma = 0.0774$

Actuation  
Power  
(nondim.)

### Flap = 18%R Actuation Power



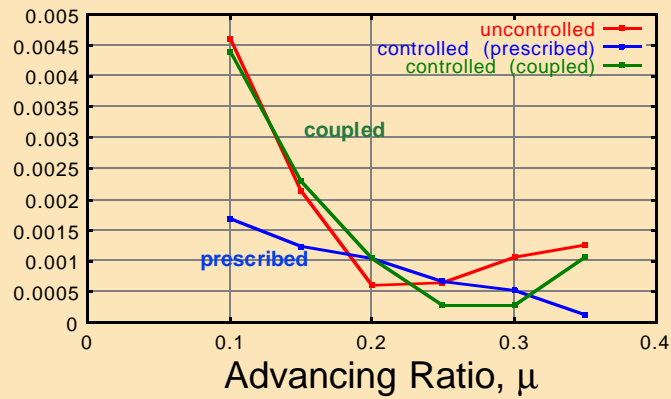
## Forward Speed Effect



- $l_f/R = 18\%$
- $c_f/c = 25\%$
- $r_f/R = 83\%$
- $C_T/\sigma = 0.0774$

### 5/rev Hub Axial Force

Axial Force  
(nondim.)



- Small reduction or small increment for axial force because of its small weight in objective function

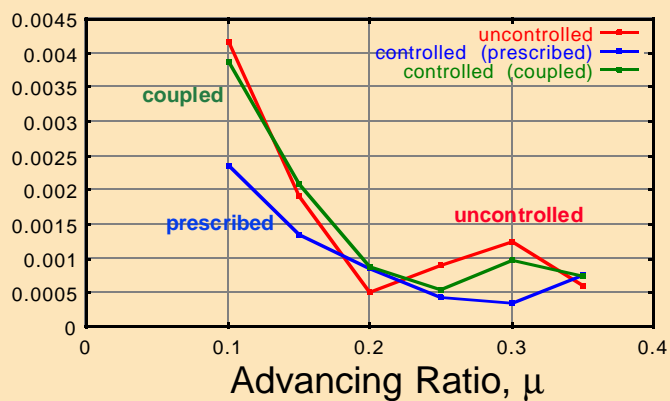
## Forward Speed Effect



- $l_f/R = 18\%$
- $c_f/c = 25\%$
- $r_f/R = 83\%$
- $C_T/\sigma = 0.0774$

### 5/rev Hub Side Force

Side Force  
(nondim.)



- Small reduction or small increment for side force because of its small weight in objective function

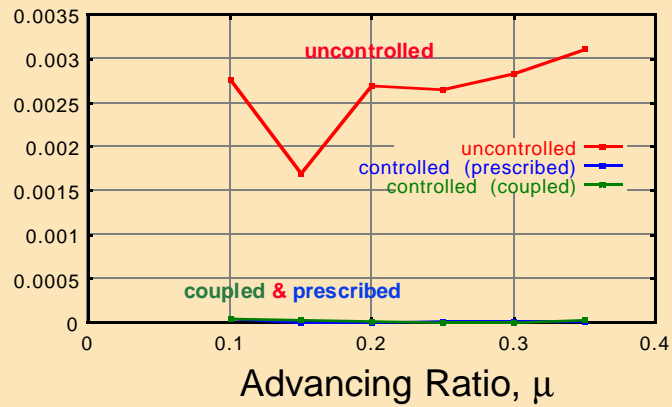
## Forward Speed Effect



- $I_{\dot{\varphi}}/R = 18\%$
- $c_{\dot{\varphi}}/c = 25\%$
- $r_{\dot{\varphi}}/R = 83\%$
- $C_T/\sigma = 0.0774$

Vertical Force  
(nondim.)

### 5/rev Hub Vertical Force



- Complete reduction for vertical force because of its dominant weight in objective function

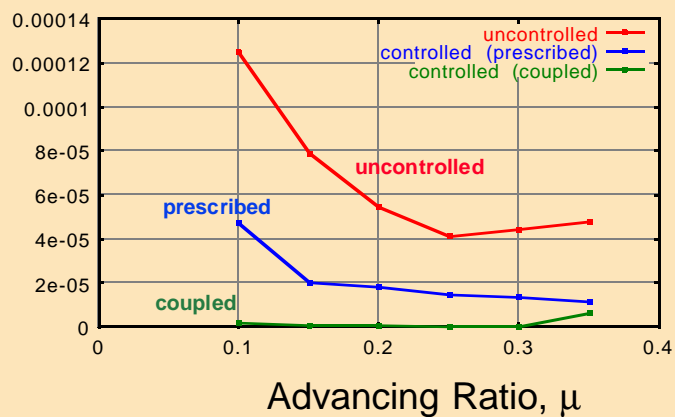
## Forward Speed Effect



- $I_{\dot{\varphi}}/R = 18\%$
- $c_{\dot{\varphi}}/c = 25\%$
- $r_{\dot{\varphi}}/R = 83\%$
- $C_T/\sigma = 0.0774$

Rolling Moment  
(nondim.)

### 5/rev Hub Rolling Moment



- For prescribed system, rolling moment reduced to 40%
- For coupled system, rolling moment reduced to 1% to 5%

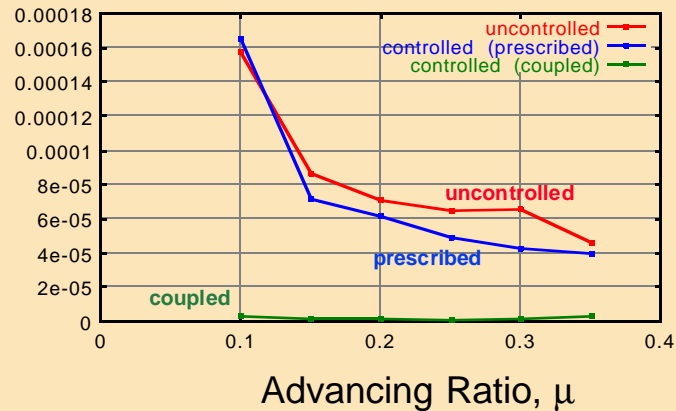
## Forward Speed Effect



- $l_f/R = 18\%$
- $c_f/c = 25\%$
- $r_f/R = 83\%$
- $C_T/\sigma = 0.0774$

### 5/rev Hub Pitching Moment

Pitching Moment  
(nondim.)



- For prescribed system, rolling moment reduced to 90%
- For coupled system, rolling moment reduced to 1% to 3%

## Summary



- Coupled blade-flap-actuator modeling was formulated .
- Coupling effect due to actuator dynamics on blade response and loads was examined.
- Correlation with CAMRAD II predictions was fair.
- Parametric study with coupled blade-flap-actuator system was carried out.

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## Conclusions



- Flap/Actuator dynamics increases the vibratory hub loads if flap is unlocked in uncontrolled condition.
  - 5/Rev hub forces increase 50 %
  - 5/Rev hub moments increase 400%
- The optimal flap location, size and pitch link stiffness are same for both coupled and prescribed system, however, the required flap peak-to-peak deflection and actuation power increase due to flap/actuator dynamics.
  - flap peak-to-peak deflection increases 90% to 200%
  - flap actuation power increases 30% to 40%
- Vibration objective function can be minimized to less than 10% for advancing ratios from 0.1 to 0.35